# ELECTRIC COMPRESSOR

## BACKGROUND OF THE INVENTION

The present invention relates to an electric compressor used in a vehicle air conditioner.

A typical electric scroll compressor used in a vehicle air conditioner has a stationary scroll and a movable scroll. The stationary scroll is fixed to a housing, and has a base plate and a volute portion. The movable scroll has a base plate and a volute portion. The volute portions intermesh. When an electric motor accommodated in the housing is driven and the movable scroll orbits, each of compression chambers defined between the volute portions is moved toward the center of the volute portions, while the volume of the compression chamber is progressively decreased. Accordingly, refrigerant gas is compressed.

Japanese Laid-Open Patent Publication No. 2002-295369 discloses an electric scroll compressor that lubricates an orbiting mechanism that permits a movable scroll to orbit relative to a stationary scroll. The scroll compressor of the publication also improves the sealing property of compression chambers against a compression reaction force in a thrust direction applied to the movable scroll. Specifically, the scroll compressor has a back pressure chamber at the back side of the base plate of the movable scroll. The back pressure chamber surrounds the orbiting mechanism. Lubricating oil the pressure of which corresponds to a discharge pressure is retained in a bottom portion of a discharge chamber. The lubricating oil is guided to the back pressure chamber so that the movable scroll is urged toward the stationary scroll. Accordingly, the sealing property of the compression chambers is

improved.

In the electric scroll compressor of the publication, lubricating oil that lubricates the orbiting mechanism and increases the back pressure falls by the self weight down to a motor accommodating chamber through an oil bleed passage having a constriction. The lubricating oil is then temporarily retained in a reservoir formed in the bottom of the motor accommodating chamber. Thereafter, the lubricating oil is sent to a suction side of the compression mechanism, which includes the volute portions of the stationary scroll and the movable scroll, through a conveying passage.

When used in a vehicle air conditioner, the above described electric scroll compressor has the following drawbacks. The reservoir for lubricating oil is formed in the bottom of the motor accommodating chamber. Therefore, when a significant amount of liquid refrigerant returns to the compressor from a refrigeration circuit, mixture of the lubricating oil and the liquid refrigerant stays in the lubricating oil reservoir. The coils of the motor and other components can be impregnated with the mixture. In a typical electric compressor, polyol ester (POE) is used as lubricating oil, so that the lubricating oil exerts a sufficient insulating performance even if mixed with liquid refrigerant. An electric compressor using such lubricant oil has no drawbacks when applied to an ordinary air conditioner.

However, in vehicle air conditioners, polyalkylene glycol (PAG) is predominantly used as lubricating oil for belt driven compressors. When mixed with liquid refrigerant, PAG significantly degrades the insulating property of the mixture liquid. When performing maintenance

of such a vehicle air conditioner, PAG can be mixed with liquid refrigerant. If wire connections and stator coils are impregnated with such mixture of the lowered insulating property, leakage of electricity can occur.

Such leakage of electricity can occur not only in electric scroll compressors, but also in electric swash plate type compressors and electric vane compressors.

## SUMMARY OF THE INVENTION

Accordingly, it is an objective of the present invention to provide an electric compressor that prevents mixture of liquids having a lowered insulating property from staying in a motor accommodating chamber.

To achieve the above-mentioned objective, the present invention provides an electric compressor. The compressor includes an electric motor having an axis of rotation and a compression mechanism that is driven by the electric motor to compress gas. The compression mechanism includes a suction chamber. A housing accommodates the compression mechanism. The housing defines a motor accommodating chamber that accommodates the electric motor such that the rotation axis of the motor is substantially horizontal. The pressure in the motor accommodating chamber is equal to the pressure in the suction chamber. A connecting passage connects a bottom portion of the motor accommodating chamber with the suction chamber.

Other aspects and advantages of the invention will become apparent from the following description, taken in conjunction with the accompanying drawings, illustrating by way of example the principles of the invention.

## BRIEF DESCRIPTION OF THE DRAWINGS

The invention, together with objects and advantages thereof, may best be understood by reference to the following description of the presently preferred embodiments together with the accompanying drawings in which:

Fig. 1 is a longitudinal cross-sectional view illustrating an electric scroll compressor according to the present invention;

Fig. 2 is a transverse cross-sectional view illustrating a compression mechanism of the compressor shown in Fig. 1;

Fig. 3 is a transverse cross-sectional view illustrating a discharge chamber of the compressor shown in Fig. 1;

Fig. 4 is an enlarged longitudinal cross-sectional view illustrating a section including an elastic body of the compressor shown in Fig. 1; and

Fig. 5 is an exploded perspective view illustrating the shaft supporting member, the elastic body, and the stationary scroll of the compressor shown in Fig. 1.

## DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

In the drawings, like numerals are used for like elements throughout.

One embodiment of the present invention will now be described with reference to the drawings.

As shown in Fig. 1, an electric scroll compressor used in a vehicle air conditioner has a compressor housing 11. The housing 11 is formed of a first housing member 12 and a second housing member 13, which are aluminum alloy castings

fastened to each other with bolts. The first housing member 12 is shaped like a horizontally oriented cylinder and includes a large diameter portion 12a, a small diameter portion 12b, and an end wall 12c. The small diameter portion 12b is integrally formed with the large diameter portion 12a at the left end of the large diameter portion 12a. The end wall 12c is integrally formed with the left end of the small diameter portion 12b, thereby closing the left end of the portion 12b. The second housing member 13 is shaped like a horizontally oriented cylinder with one end closed. A sealed space 14 is defined in the housing 11. The sealed space 14 is encompassed by the housing members 12, 13.

A cylindrical shaft supporting portion 12d extends from a center portion of the inner surface of the end wall 12c, which is a part of the first housing member 12. A shaft supporting member 15 is fitted and fixed to an open end of the large diameter portion 12a of the first housing member 12. The shaft supporting member 15 functions as a partition member, or a stationary wall, and has a through hole 15a in the center. A rotary shaft 16 is accommodated in the first housing member 12. The left end of the rotary shaft 16 is rotatably supported by the shaft supporting portion 12d with a bearing 17 in between. The right end of the rotary shaft 16 is rotatably supported by the through hole 15a of the shaft supporting member 15 with the bearing 18 in between. A sealing member 19 is located between the shaft supporting member 15 and the rotary shaft 16 to seal the rotary shaft 16. Accordingly, a motor accommodating chamber 20 is defined in a left portion of the sealed space 14 as viewed in Fig 1. The shaft supporting member 15 is a wall of the motor accommodating chamber 20.

In the motor accommodating chamber 20, a stator 21

having a coil 21a is located on the inner surface of the small diameter portion 12b of the first housing member 12. In the motor accommodating chamber 20, a rotor 22 is fixed to the rotary shaft 16. The rotor 22 is located radially inward of the stator 21. The small diameter portion 12b, the shaft supporting member 15, the rotary shaft 16, the stator 21, and the rotor 22 form an electric motor 23. An axis of rotation of the motor 23 extends horizontally. The rotation axis coincides with an axis L of the rotary shaft 16. When electricity is supplied to the coil 21a of the stator 21, the rotary shaft 16 and the rotor 22 rotate integrally.

In the first housing member 12, a stationary scroll 24 is located at the open end of the large diameter portion 12a. The stationary scroll 24 includes a disk-shaped base plate 24a, a circumferential wall 24b, and a volute portion 24c. The circumferential wall 24b is integrally formed with and arranged lateral to the base plate 24a. The volute portion 24c is also integrally formed with the base plate 24a. The volute portion 24c is located on a front side (left side as viewed in Fig. 1) of the base plate 24a and inside the circumferential wall 24b (see Fig. 2). A flange portion 15b is integrally formed with the outer circumferential portion of the shaft supporting member 15. The stationary scroll 24 contacts the flange portion 15b at the distal end face of the circumferential wall 24b (see Fig. 4). Therefore, in the sealed space 14, the base plate 24a and the circumferential wall 24b of the stationary scroll 24, the shaft supporting member 15, and the sealing member 19 sealing the rotary shaft 16 define a scroll accommodating chamber 25 between the shaft supporting member 15 and the stationary scroll 24.

An eccentric shaft 26 is located at the distal end face

of the rotary shaft 16. The eccentric shaft 26 is displaced from the axis L of the rotary shaft 16 and is located in the scroll accommodating chamber 25. A bushing 27 is fitted and fixed to the eccentric shaft 26. A movable scroll 28 is accommodated in the scroll accommodating chamber 25. The movable scroll 28 is rotatably supported by the bushing 27 with a bearing 29 in between such that the movable scroll 28 faces the stationary scroll 24. The movable scroll 28 includes a disk-shaped base plate 28a and a movable volute portion 28b. The base plate 28a includes a first face, or a front face (right end face as viewed in Fig. 1) and a second face, or a back face (left end face as viewed in Fig. 1). The movable volute portion 28b extends from the first face, and the second face is opposite from the first face. The movable volute portion 28b is integrally formed with the front face of the base plate 28a. As shown in Fig. 4, an annular projection 28c, which is annular when viewed along a thrust direction, is integrally formed with the base plate 28a on the peripheral portion. The annular projection 28c faces the flange portion 15b. The surface of the movable scroll 28 is plated with nickel phosphorus (Ni-P).

The stationary scroll 24 and the movable scroll 28 intermesh at the volute portions 24c, 28b in the scroll accommodating chamber 25. The distal end face of each of the volute portions 24c, 28b contacts the base plate 28a, 24a of the other scroll 28, 24. Therefore, the base plate 24a and the stationary volute portion 24c of the stationary scroll 24 and the base plate 28a and the movable volute portion 28b of the movable scroll 28 define a compression chamber 30 in the scroll accommodating chamber 25.

Anti-rotation mechanism 31 is provided between the base plate 28a of the movable scroll 28 and the shaft supporting member 15, which faces the base plate 28a. The anti-

rotation mechanism 31 includes circular holes 28d formed in the peripheral portion of the back of the base plate 28a of the movable scroll 28 and pins 32 (only one is shown in the drawing) projecting from the flange portion 15b of the shaft supporting member 15. The pins 32 are loosely fitted in the circular holes 28d.

In the scroll accommodating chamber 25, a suction chamber 33 is defined between the circumferential wall 24b of the stationary scroll 24 and the outermost portion of the movable volute portion 28b of the movable scroll 28. In a lower portion of the circumferential wall 24b of the stationary scroll 24, symmetric two recesses 24d are formed as shown in Figs. 2, 3 and 5. In an inner lower surface of the large diameter portion 12a of the first housing member 12, symmetrical two recess 12e are formed to correspond to the recesses 24d. A space between the inner surfaces of the recesses 12e and the outer surface of the flange portion 15b of the shaft supporting member 15, and the recesses 24d of the circumferential wall 24b define a connecting passage 34 that connects a bottom portion, which is the lowest portion of the motor accommodating chamber 20 with the suction chamber 33.

That is, the connecting passage 34 is formed by denting a portion of the inner surface of the first housing member 12 that faces the outer surface of the stationary scroll 24. The connecting passage 34 extends between the inner surface of the first housing member 12 and the outer surface of the stationary scroll 24. The connecting passage 34 extends horizontally for a certain length from the bottom portion of the motor accommodating chamber 20 toward a lower portion of the suction chamber 33, and then extends upward toward the suction chamber 33. The lowest portion of the inner surface of the recess 12e, that is, the lowest section of a face

defining the connecting passage 34 is located lower than the lowest part of the motor 23.

As shown in Fig. 1, in a left outer portion of the small diameter portion 12b of the first housing member 12 as viewed in Fig. 1, a suction port 12f is formed to permit the motor accommodating chamber 20 to communicate with the outside. An external pipe is connected to the suction port 12f. The external pipe is connected to an evaporator of an external refrigerant circuit (not shown). Therefore, low pressure refrigerant gas is drawn into the suction chamber 33 from the external refrigerant circuit through the suction port 12f, the motor accommodating chamber 20 and the connecting passage 34. The suction port 12f, the motor accommodating chamber 20 and the connecting passage 34 form a suction passage. Although not illustrated, grooves extending in a thrust direction are formed on the outer circumferential surface of the stator 21. The grooves function as passages for refrigerant gas.

A discharge chamber 35 is defined between the second housing member 13 and the stationary scroll 24. A discharge hole 24e is formed in a center portion of the base plate 24a of the stationary scroll 24. The discharge hole 24e connects the compression chamber 30 with the discharge chamber 35 when the compression chamber 30 is at the center of the scrolls 24, 28. In the discharge chamber 35, a discharge valve 37, which is a reed valve, is provided on the stationary scroll 24 to open and close the discharge hole 24e. The opening degree of the discharge valve 37 is limited by a retainer 38 fixed to the stationary scroll 24. A discharge port 13a is formed in the second housing member 13. The discharge port 13a communicates with the discharge chamber 35. An external pipe is connected to the discharge port 13a. The external pipe is connected to a cooler of the

external refrigerant circuit (not shown). An oil separator 36 is attached to the discharge port 13a to separate lubricating oil from high pressure refrigerant gas. Therefore, high pressure refrigerant gas in the discharge chamber 35 is discharged to the external refrigerant circuit through the discharge port 13a after the oil separator separates lubricating oil from the refrigerant gas. A first reservoir chamber 39 is formed in a bottom portion of the discharge chamber 35 to retain lubricating oil that has been separated from refrigerant by the oil separator 36.

When the rotary shaft 16 is rotated by the electric motor 23, the movable scroll 28 is caused to orbit about the axis (the axis L of the rotary shaft 16) by the eccentric shaft 26. The axis of the stationary scroll 24 coincides with the axis L of the rotary shaft L. The movable scroll 28 is prevented from rotating by the anti-rotation mechanism 31, but is only permitted to orbit. The orbiting motion of the movable scroll 28 moves the compression chamber 30 from an outer portion of the volute portions 24c, 28b of the scrolls 24, 28 toward the center while decreasing the volume of the compression chamber 30. Accordingly, low pressure refrigerant that has been drawn into the compression chamber 30 from the suction chamber 33 is compressed. The compressed high pressure refrigerant gas is discharged to the discharge chamber 35 through the discharge hole 24e while opening the discharge valve 37.

As shown in Figs. 1 and 4, a back pressure chamber 41 is defined in the scroll accommodating chamber 25 at the back of the base plate 28a of the movable scroll 28. The back pressure chamber 41 and the first reservoir chamber 39, which is located in a lower portion of the discharge chamber 35, or a discharge pressure zone, are connected with each other by a pressurized oil supply passage 42. The

pressurized oil supply passage 42 has a constriction 42a (see Fig. 5). The high pressure lubricating oil containing a small amount of refrigerant gas is supplied to the back pressure chamber 41 from the first reservoir chamber 39 at a bottom portion of the discharge chamber 35 and urges the movable scroll 28 toward the stationary scroll 24.

As shown in Figs. 1, 4 and 5, in the scroll accommodating chamber 25, an elastic body 51, which is a doughnut-shaped plate, is located between the flange portion 15b of the shaft supporting member 15 and the circumferential wall 24b of the stationary scroll 24. The elastic body 51 is made, for example, of metal such as carbon steel. A peripheral portion of the elastic body 51 is held between the flange portion 15b of the shaft supporting member 15 and the circumferential wall 24b of the stationary scroll 24, so that the elastic body 51 is fixed in the scroll accommodating chamber 25.

As shown in Fig. 5, an arcuate elongated hole 51a is formed in a peripheral portion of the elastic body 51. The elongated hole 51a and a space encompassed by a contact surface 15c of the flange portion 15b of the shaft supporting member 15 and a distal end face of the circumferential wall 24b of the stationary scroll 24 form a section (constriction 42a) of the pressurized oil supply passage 42 connecting the first reservoir chamber 39 with the back pressure chamber 41. The lower end of the elongated hole 51a is connected with the first reservoir chamber 39 by an oil passage 24f formed in the circumferential wall 24b of the stationary scroll 24. The upper end of the elongated hole 51a is connected with the back pressure chamber 41 by a wide annular groove 15d and a linear groove 15e, which are formed in the contact surface

15c of the shaft supporting member 15. The oil passage 24f, the elongated hole 51a, and the grooves 15d, 15e form the pressurized oil supply passage 42.

As shown in Fig. 4, the elastic body 51 is installed while being elastically deformed by the annular projection 28c of the movable scroll 28. The elasticity of the elastic body 51 maintains the sealing property between the elastic body 51 and the contact surface of the annular projection 28c, and urges the movable scroll 28 toward the stationary scroll 24. Therefore, the elastic body 51 and the annular projection 28c seal the back pressure chamber 41 and the suction chamber 33 from each other.

Fig. 3 illustrates a state where the second housing member 13 is removed from the open end of the large diameter portion 12a of the first housing member 12. As shown in Figs. 1 and 3, a dividing wall 24g, which is shaped like a closed ring, is integrally formed with the base plate 24a of the stationary scroll 24. The dividing wall 24g projects from the back of the base plate 24a. A dividing wall 13b, which corresponds to the dividing wall 24g, is integrally formed with the second housing member 13 on an inner surface. As shown in Fig. 3, an accommodating groove m is formed in the distal end face of the dividing wall 24g. A seal ring 52 is fitted in the groove m to seal the distal end face of the dividing wall 13b. As shown in Figs. 1 and 3, the discharge chamber 35 is defined inward of the dividing walls 24g, 13b. A second reservoir chamber 53 is defined between the circumferential surfaces of the dividing walls 24g, 13b and the inner surface of the second housing member 13. The second reservoir chamber 53 and the back pressure chamber 41 are connected with each other by an oil bleed passage 54 formed in the flange portion 15b of the shaft supporting member 15 and the circumferential wall 24b

of the stationary scroll 24. As shown in Fig. 5, the oil bleed passage 54 includes a recess 15f, a hole 51b, and a passage 24h. The recess 15f is formed in the contact surface 15c of the shaft supporting member 15 and communicates with the groove 15d. The hole 51b extends through a peripheral portion of the elastic body 51 and corresponds to the recess 15f. The passage 24h is formed in the circumferential wall 24b of the stationary scroll 24 to correspond to the hole 51b. Pin holes 51c are formed in an inner portion of the elastic body 51. The pins 32 of the anti-rotation mechanism 31 are inserted in the pin holes 51c.

As shown in Fig. 1, an adjuster valve 55 is located in a section of the oil bleed passage 54, or a section of the passage 24h, in the circumferential wall 24b of the stationary scroll 24. The adjuster valve 55 adjusts the opening degree of the oil bleed passage 54 according to the difference between the pressure in the back pressure chamber 41 and the pressure in the second reservoir chamber 53. The adjuster valve 55 includes a ball valve 56 and a coil spring 57, and operates to maintain the pressure difference between the back pressure chamber 41 and the second reservoir chamber 53 to a constant value. Therefore, when the electric scroll compressor operates normally, the adjuster valve 55 maintains the pressure in the back pressure chamber 41, or an urging force of the movable scroll 28 based on the pressure in the back pressure chamber 41, to a constant value. Further, lubricating oil in the back pressure chamber 41 is sent to the second reservoir chamber 53 through the oil bleed passage 54 and the adjuster valve 55 and retained in the second reservoir chamber 53.

As shown in Fig. 3, an oil return passage 24i is formed in the base plate 24a of the stationary scroll 24. The oil

return passage 24i connects the bottom of the second reservoir chamber 53 with the suction chamber 33. A gas return passage 24j is formed in the base plate 24a to connect an upper portion of the second reservoir chamber 53 with an upper portion of the suction chamber 33. The gas return passage 24j returns gas separated from lubricating oil retained in the second reservoir chamber 53 to the suction chamber 33. Therefore, lubricating oil retained in the second reservoir chamber 53 is drawn to the suction chamber 33 through the oil return passage 24i by a suction effect based on orbiting motion of the movable scroll 28. The lubricating oil is then drawn into the compression chamber 30 with refrigerant gas to lubricate sliding surfaces of the compression mechanism. Further, refrigerant gas separated from lubricating oil stays in an upper portion of the second reservoir chamber 53 and is returned to the suction chamber 33 through the gas return passage 24j.

Since the recesses 24d forming the connecting passage 34 is formed in the base plate 24a as shown in Fig. 3, the shape of the outer contact surface of the second housing member 13 is determined to define the recesses 24d and the second reservoir chamber 53. As shown by alternate long and two short dashes lines in Fig. 3, a partition gasket 58 is located between the outer contact surface and the open end face of the large diameter portion 12a of the first housing member 12.

As shown in Fig. 1, an accommodating recess 61 is formed by bulging a bottom portion of the large diameter portion 12a of the first housing member 12 downward. The accommodating recess 61 is capable of retaining a predetermined amount of lubricating oil and liquid refrigerant below the coil 21a.

The illustrated embodiment provides the following advantages.

(1) In the illustrated embodiment, the electric motor 23 is mounted horizontally in the motor accommodating chamber 20 defined in the first housing member 12. The motor accommodating chamber 20 functions as a part of the suction passage of refrigerant gas. Refrigerant gas is drawn into the suction chamber 33 from the bottom portion of the motor accommodating chamber 20 through the connecting passage 34. Thus, during a normal operation of the compressor, lubricating oil and liquid refrigerant in a bottom portion of the motor accommodating chamber 20 are drawn into the suction chamber 33 together with suction refrigerant gas, and are prevented from staying in the motor accommodating chamber 20. In a case where POE lubricating oil and PAG lubricating oil are used together and the mixed lubricating oil is mixed with liquid refrigerant, the mixed liquid has a lowered insulating property. The illustrated embodiment prevents the coil 21a of the electric motor 23 from being impregnated with the such mixed liquid. As a result, leakage of electricity is prevented.

(2) In the illustrated embodiment, the accommodating recess 61 is formed in a lower part of the large diameter portion 12a of the first housing member 12, which lower part is located below the stator 21. In other words, the accommodating recess 61 is located lower than the motor 23. In the interior of the motor accommodating chamber 20, when the compressor is temporarily stopped, lubricating oil contained in refrigerant gas can be retained in a bottom portion of the motor accommodating chamber 20 due to the physical property of the air conditioner. Even if this is the case, the illustrated embodiment prevents the coil 21a of the stator 21 from being impregnated with the mixed

liquid of a lowered insulating property. Therefore, when the compressor is started again, leakage of electricity is prevented.

(3) In the illustrated embodiment, the discharge chamber 35 is defined between the second housing member 13 and the base plate 24a of the stationary scroll 24. The second reservoir chamber 53 is defined outside of the discharge chamber 35. Lubricating oil is supplied to the second reservoir chamber 53 from the back pressure chamber 41 through the oil bleed passage 54 and the adjuster valve 55, and is temporarily retained in the second reservoir chamber 53.

Further, lubricating oil is supplied to the suction chamber 33 from the second reservoir chamber 53 through the oil return passage 24i. Therefore, lubricating oil is reliably supplied to the suction chamber 33 from the second reservoir chamber 53. This reliably lubricates the sliding surfaces of the compression mechanism.

In the illustrated embodiment, a part of the suction chamber (low pressure zone), which is conventionally given no additional functions, is used as the second reservoir chamber 53. Therefore, there is no need for providing dedicated components for the second reservoir chamber 53. This reduces the manufacturing cost.

(4) The movable scroll 28 is urged toward the stationary scroll 24 by high pressure refrigerant gas supplied to the back pressure chamber 41. That is, the movable scroll 28 is urged toward the stationary scroll 24 not only by the urging force generated by elastic deformation of the elastic body 51, but also by the urging force generated by the pressure of the back pressure chamber

41. These urging forces reliably act against the compression reaction force in the thrust direction acting on the movable scroll 28 during a normal operation of the electric compressor. Thus, in the illustrated embodiment, in which sealing members (for example, chip seals) are not provided on the end faces of the volute portions 24c, 28b, the compression chamber 30 is reliably sealed.

(5) The surface of the movable scroll 28 is plated with nickel phosphorus (Ni-P). When a high-speed operation of the compressor is continued, lubrication will be insufficient in the compressor. Even if this is the case, the plated surface of the movable scroll 28 increases the durability of the sliding surfaces of the stationary scroll 24 and the movable scroll 28.

The invention may be embodied in the following forms.

The suction port 12f of the first housing member 12 may be omitted so that the motor accommodating chamber 20 does not function as a part of the suction passage, and the suction port 12f may be formed in the bottom of the large diameter portion 12a. Also in this case, the recess 12e functions as a connecting passage that connects the bottom portion of the motor accommodating chamber 20 with the suction chamber 33 of the compression mechanism.

In this modified embodiment, liquid refrigerant does not return to the motor accommodating chamber 20 from the refrigeration circuit. Therefore, no mixture of liquid refrigerant and other kinds of lubricating oils is generated in the motor accommodating chamber 20. Leakage of electricity at the wire joints and the coil 21a of the electric motor 23 is thus prevented.

In the illustrated embodiment, the recess 12e may be omitted, and the connecting passage may be formed in the flange portion 15b of the shaft supporting member 15 and a lower portion of the circumferential portion of the elastic body 51. This connecting passage may be formed as a groove or a through hole.

In the illustrated embodiment, the adjuster valve 55 in the oil bleed passage 54 may be replaced by a constriction having a smaller cross-sectional area than the constriction 42a.

In the illustrated embodiment, the rotation axis L of the electric motor 23 is arranged horizontally. However, as long as the rotation axis L is substantially horizontal, the axis L may be inclined upward or downward, for example, by 10° relative to a horizontal line.

In the illustrated embodiment, the present invention is applied to an electric scroll compressor. However, the present invention may be applied to any type of electric compressors such as electric swash plate type compressor, an electric vane compressor, and an electric piston compressor. Alternatively, the present invention may be applied to any type of hybrid compressors, which use an electric motor and an engine as drive sources.

The present examples and embodiments are to be considered as illustrative and not restrictive and the invention is not to be limited to the details given herein, but may be modified within the scope and equivalence of the appended claims.